

HEAT
PUMPS
IN
SPAIN

NATIONAL WORKSHOP

11TH NOVEMBER 2024

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Ongoing
Spanish
Team activity
on HPT

Evaluation of lower-GWP refrigerants for residential heat pumps

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Research from different national projects

- Liquid-to-water heat pumps with lower-GWP refrigerants
 - Funded by the Spanish 'Agencia Estatal de Investigación', within the 'Programa Estatal de I+D+i Orientada a los Retos de la Sociedad del Plan Estatal de Investigación Científica y Técnica y de Innovación 2013–20160, grant number ENE2016–79771–R
- Air-to-water heat pumps with lower-GWP refrigerants
 - Funded by the Spanish 'Agencia Estatal de Investigación', within the 'Programa Estatal de I+D+i Orientada a los Retos de la Sociedad del Plan Estatal de Investigación Científica y Técnica y de Innovación 2017–2020, grant number PID2019-104762RB-I00/AEI/10.13039/501100011033



Motivation

- Liquid-to-water heat pumps (domestic sector):
 - Variable speed compressor technology
 - Plate heat exchangers
 - No liquid receiver (dependent of refrigerant charge)
 - R410A widely used
- Global Warming Potential (GWP)
 - GWP of R410A is 1924
 - International regulations limit or ban refrigerants with high GWP (ex. EU N°517/2014)
 - Natural refrigerants
 - Blends: R452B ,R454B, R455A...
 - They show similar COP but lower capacities
 - Evaluation done at limited operating conditions



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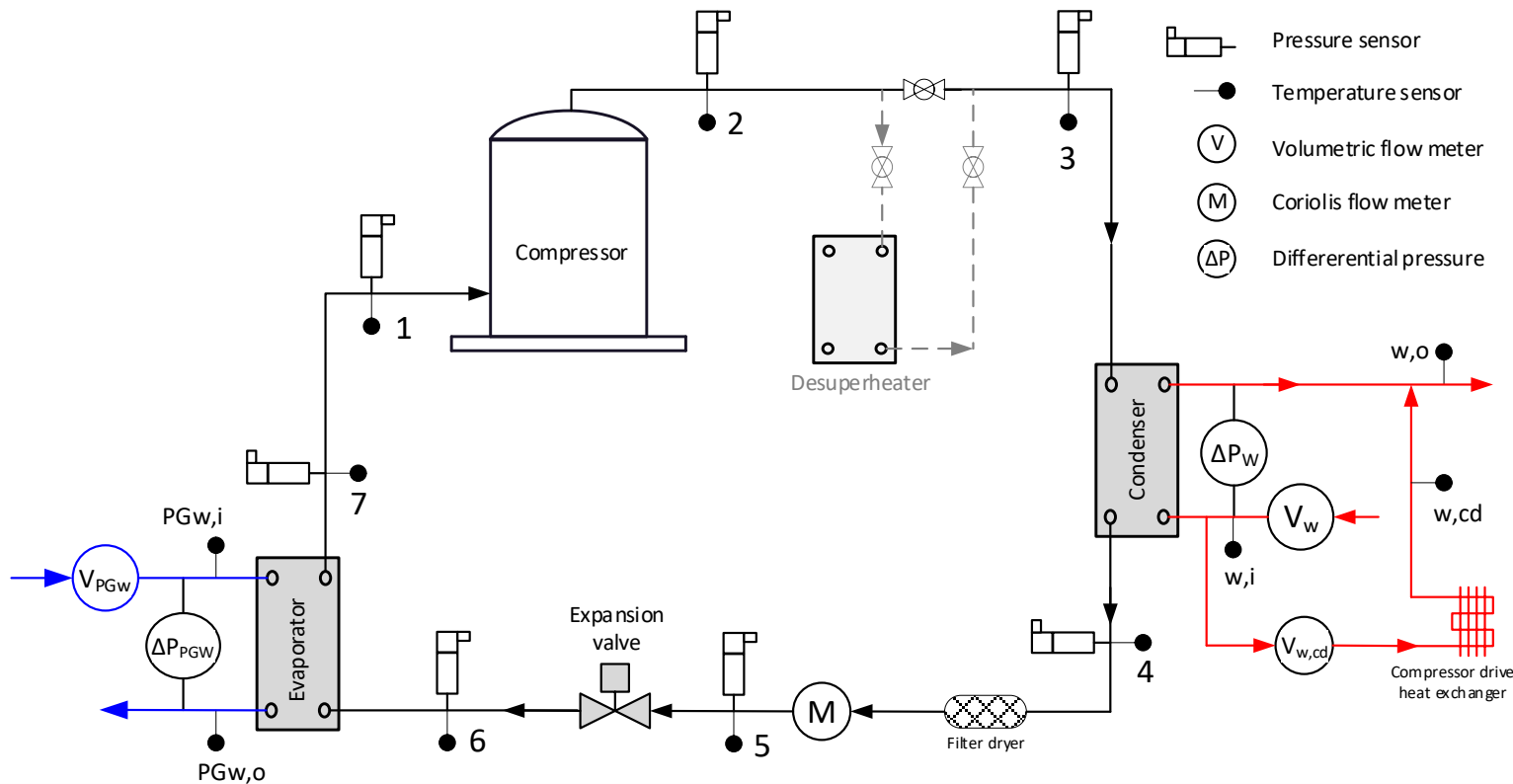
Outline

1. Refrigerants tested
2. Experimental set-up
3. Test conditions
4. Selected results
5. Conclusions and future work

Refrigerants tested

Property	R410A	R452B	R454B	R455A
Composition (wt%)	R32/R125 (50/50%)	R32/R125/R1234yf (67/7/26%)	R32/R1234yf (68.9/31.1%)	R1234yf/R32/R744 (75.5/21.5%/3%)
Molecular weight	72.6 kg/kmol	63.5 kg/kmol	62.6 kg/kmol	87.5 kg/kmol
Critical temperature	71.3 °C	77.1 °C	78.1 °C	85.6 °C
Boiling point @1 atm	-51.4 °C	-50 °C	-50.5 °C	-39.2 °C
Heat of vap. @1 atm	19813 kJ/kmol	19806 kJ/kmol	19804 kJ/kmol	20501 kJ/kmol
Density sat. liq. @1 atm	1350 kg/m ³	1015 kg/m ³	1239 kg/m ³	1033 kg/m ³
Glide	0.1 K	1.3 K	1.5 K	12.8 K
GWP (AR4)	1924	675	466	146
ASHARE Safety Class.	A1	A2L	A2L	A2L

Experimental set-up



- Liquid-to-water HP for heating and DHW production
- Variable speed scroll compressor (Nmax = 7000 rpm)
- Brazed plate HX
- Electronic expansion valve
- Refrigerant R410A

Heat pump



Propylene-glycol
water mixture tank

Water tank

DHW tank



Experimental setup - Instrumentation

Variable	Instrumentation	Accuracy
Water and PGW volumetric flow rate	Electromagnetic flow meter	±0.2% reading
Water volumetric flow rate (compressor drive HEx)	Paddle-wheel flow meter	±(0.1 L×min ⁻¹ +2.5% reading)
Water and PGW temperatures	PT 100 Class B 1/10	±0.06 K
Water and PGW pressure drops	Differential pressure transducer	± 0.1% reading
Refrigerant temperatures	PT 100 Class A	±0.4 K
Refrigerant pressures (gauge): points 2-5	Pressure transmitter	±0.04 bar
Refrigerant pressures (gauge): points 6, 7 and 1	Pressure transmitter	±0.02 bar
Refrigerant mass flow rate and density	Coriolis flow meter	±0.1% reading
Atmospheric pressure	Digital barometer	±0.003 bar
Compressor and HP auxiliary electrical power	Power transducer	±0.5% reading

Test conditions

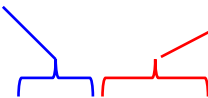
- Refrigerant charge sweeps
 - 50 grams increments (aprox.)
 - Undercharged – Overcharged conditions
 - Optimum charge determination

- Operating conditions

- PGW and water temperatures: **B0W35** and **B0W45**
- Compressor speed: 25 – 100%
- Degree of superheat: 2, 5 and 10 K

0/-3 °C (32/26.6 °F)

30/35 °C (86/95 °F)

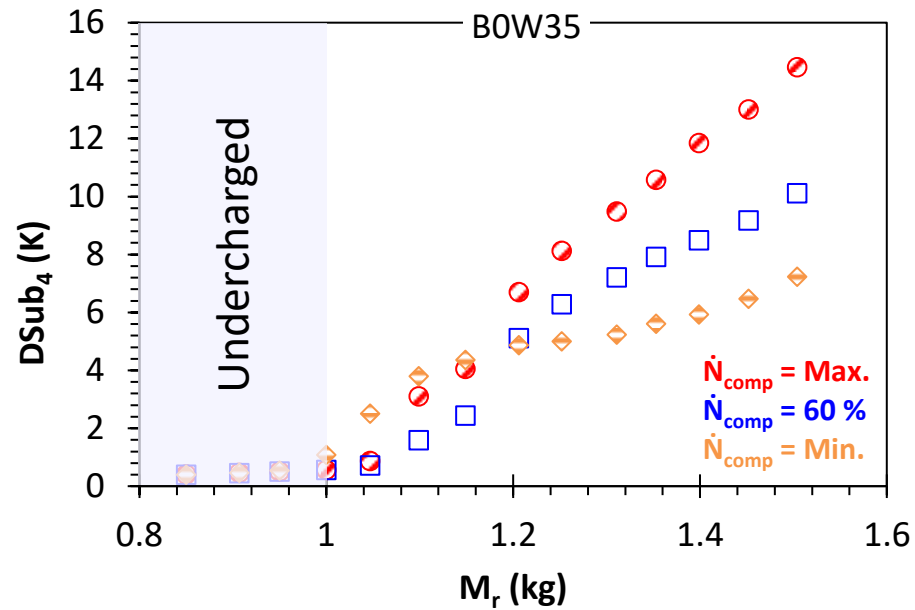


Select results – Optimum charge

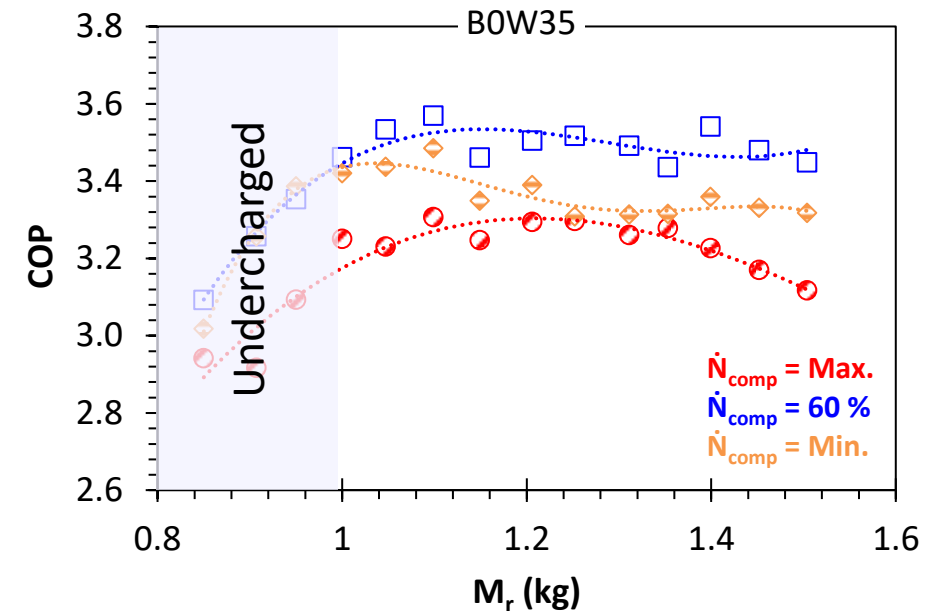
- Refrigerant: R410A (baseline)
- Optimum refrigerant charge
- Operating conditions
 - PGW inlet/outlet: 0/-3 °C (32/26.6 °F)
 - Water inlet /outlet: 30/35 °C (86/95 °F) and 40/45 °C (104/113 °F)
 - Compressor speed: 25 – 100%
 - Degree of superheat: 2, 5 and 10 K

Optimum charge – R410A

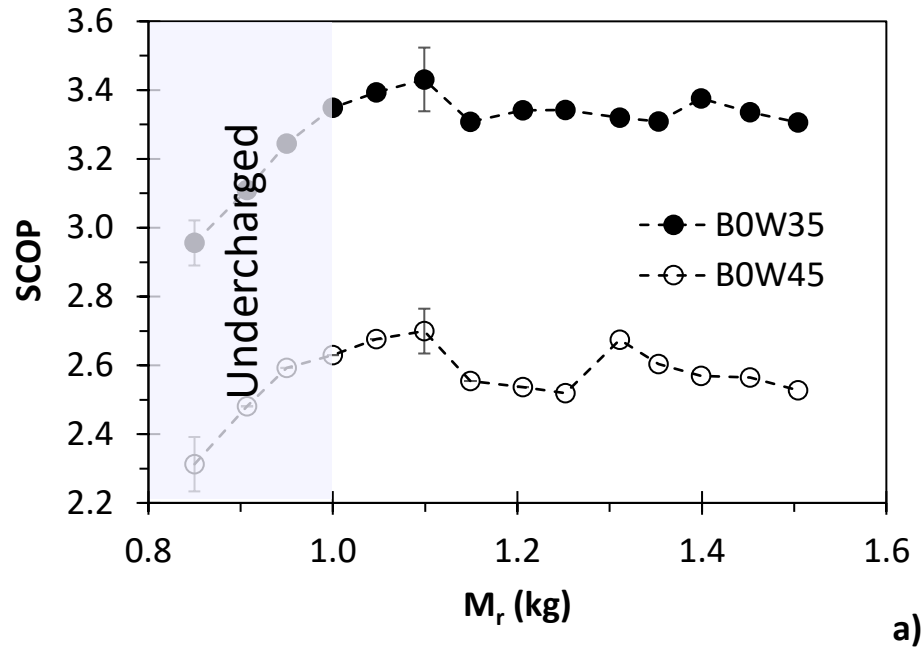
- COP drops for undercharged conditions
- COP stable if minimum charge is surpassed



- Charge that maximizes COP
- Optimum charge depends on compressor speed
- Optimum charge?

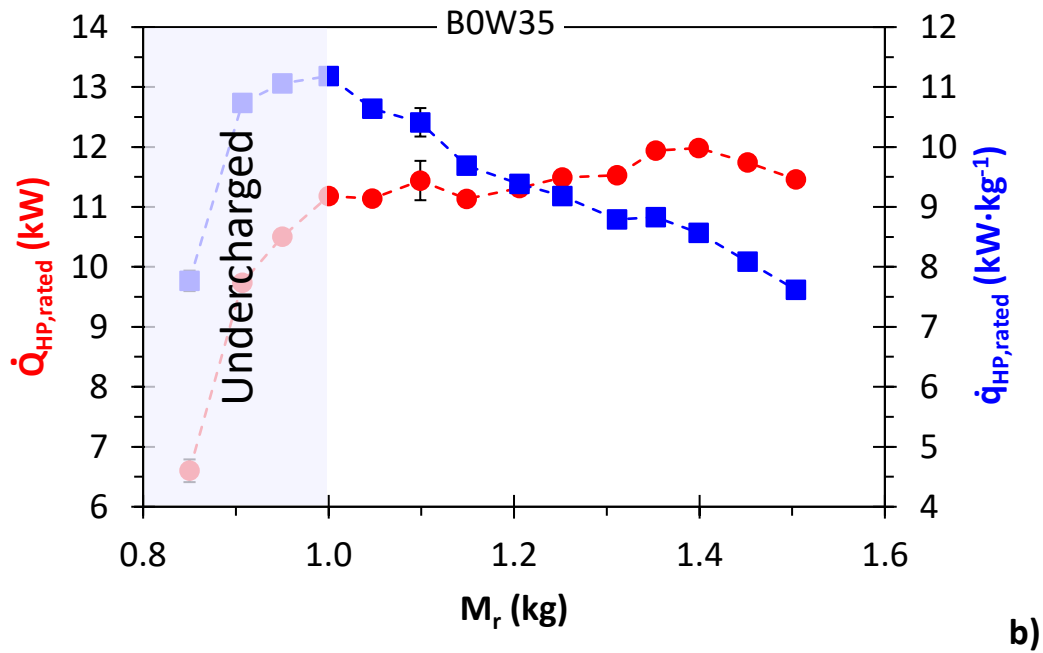


Optimum charge – R410A



- SCOP vs. charge
 - Considers the real behaviour of the unit during the heating season
- SCOP drops for undercharged conditions
- SCOP stable if minimum charge is surpassed

Optimum charge – R410A



- Rated heating capacity

- $\dot{Q}_{HP,rated} = \dot{Q}_{HP@N_{max}}$
- Drops for undercharged conditions
- Limited at high charges by compressor max. power

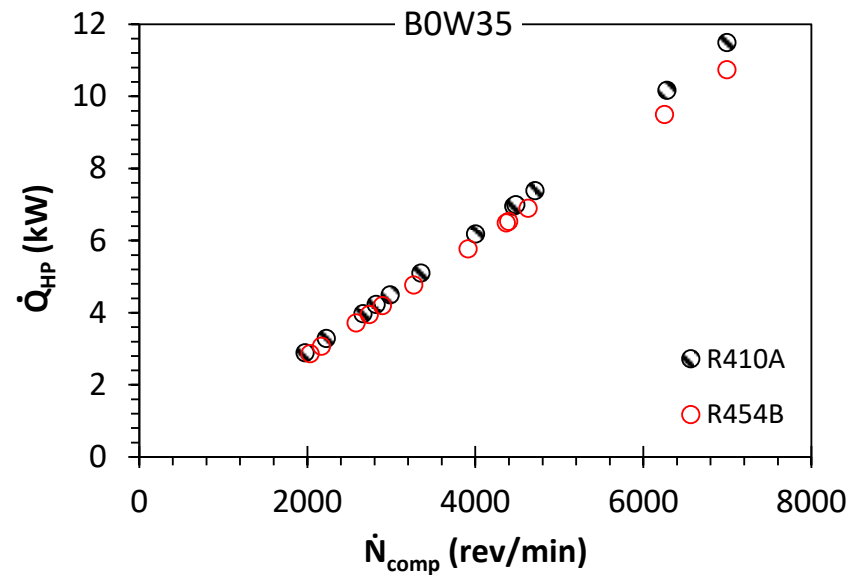
- Charge specific rated capacity

- $\dot{q}_{HP,rated} = \dot{Q}_{HP,rated} / M_r$
- Drops for undercharged conditions
- Once a minimum charge is surpassed, $\dot{q}_{HP,rated}$ decreases with an increment of the refrigerant charge

R454B vs R410A – Drop in performance

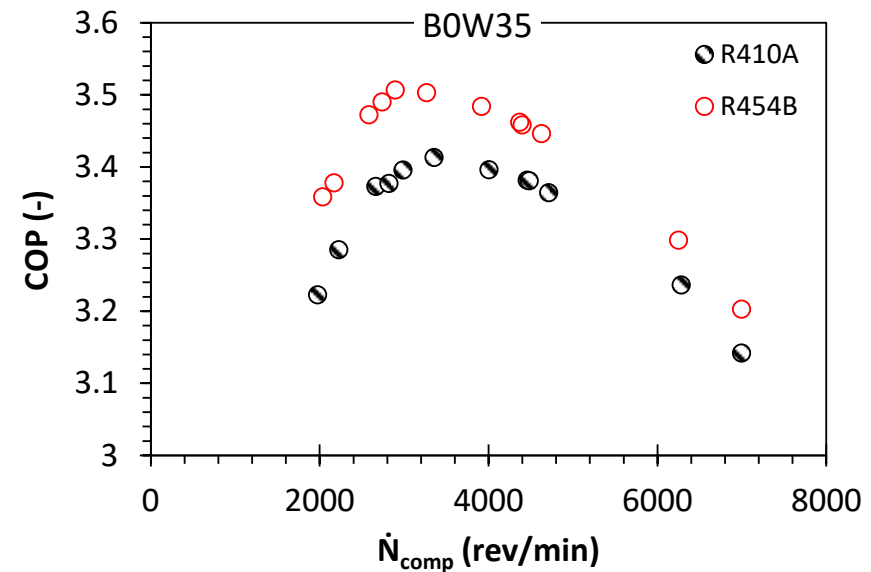
- Heating capacity

- Up to 6% lower with R454B



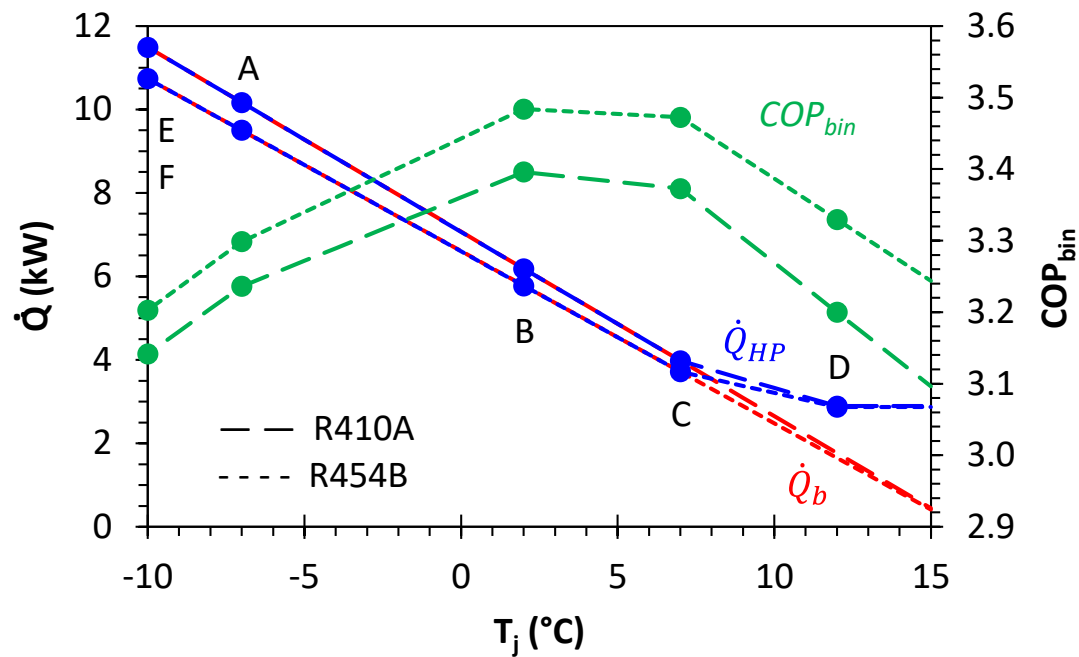
- COP

- Higher with R454B (but for lower Q_{HP})



R454B vs R410A – Drop in performance

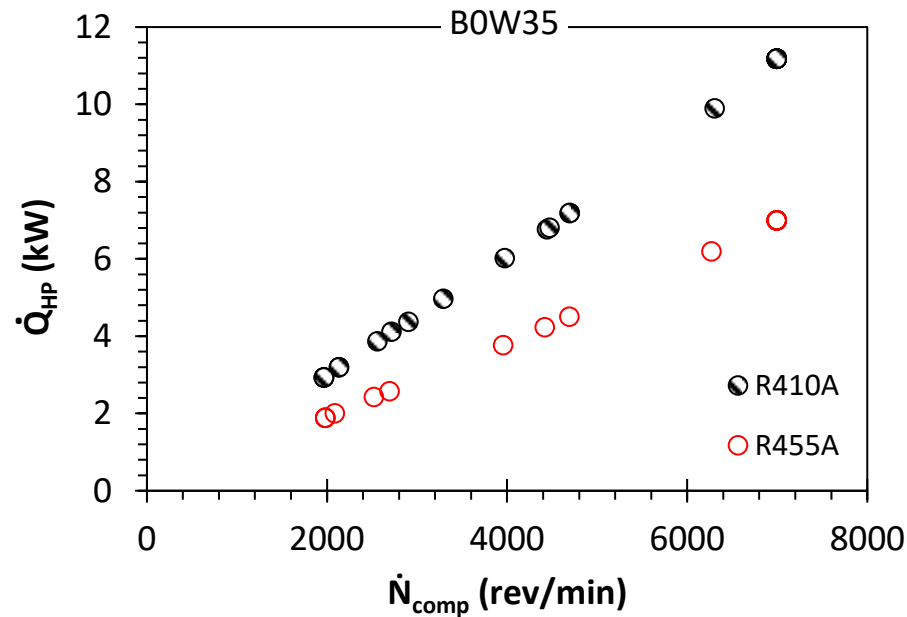
- SCOP calculation (EN 14825)



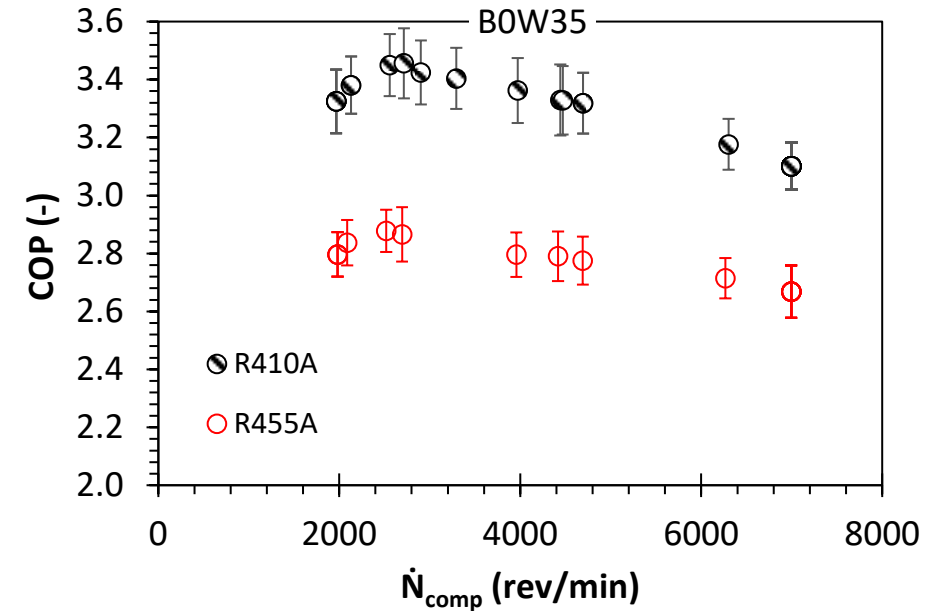
Parameter	R410A	R454B
\dot{Q}_{design} (kW)	11.49	11.49
$\dot{Q}_{HP,rated}$ (kW)	11.49	10.74
Q_b (kWh)	23736	23736
E_{on} (kWh)	7103	6949
$SCOP_{on}$	3.342	3.416

R455A vs R410A – Drop in performance

- Heating capacity
 - Up to 35% lower with R455A



- COP
 - Up to 25% lower with R455A



Conclusions and future work

- Lower-GWP refrigerants evaluation for domestic heat pumps
 - Charge optimization based on SCOP and \dot{q}_{HP}
 - R452B and R454B showed similar performance to R410A
 - R452B and R454B have GWP > 150
 - R455A has GWP < 150 but
 - Lower \dot{Q}_{HP} and COP than R410A
- Ongoing research in air-to-water HP
 - Liquid-to-water and Air-to-water HP using propane
 - Low-diameter coils for reducing refrigerant charge

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Thank you!



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